

# Thermal Protection Systems

- Types of thermal protection systems (TPS)
- Ablation
- Thermal conductivity
- Still heavily using / adapting slides from 2012 NASA Thermal and Fluids Analysis Workshop: <https://tfaws.nasa.gov/TFAWS12/Proceedings/Aerothermodynamics%20Course.pdf>
- Thermal protection slides in that package by John A. Dec / NASA Langley

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# TPS Outline

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- ◆ **Background Information**
  - What is TPS?
  - Selecting the Right Material for the Mission
- ◆ **Ablative TPS Modeling**
  - Ablator Characteristics
  - Surface Recession
  - In-Depth Models
- ◆ **TPS Sizing and Margin**
- ◆ **TPS Testing**
- ◆ **Look to the Future**



# Types of Thermal Protection Systems

- Ablative (non-reusable)
  - Carries heat away by ablation processes and pyrolysis
  - Effectively passive transpiration cooling
  - Ideal for high heat flux / load entries
- Heat Sinks (reusable)
  - Entry heating is absorbed by surface material - high specific heat, low thermal conductivity
  - Heat is ultimately dissipated by surface cooling and / or jettisoning
- Active (reusable)
  - Active cooling via fluid injection through surface into flow
  - Complex; heavy; low technology readiness levels

# 1D Conduction

- Basic law of one-dimensional heat conduction (Fourier 1822)

$$Q = -KA \frac{dT}{dx}$$

where

K=thermal conductivity (W / m°K)

A=area

dT / dx=thermal gradient

# 3D Conduction

General differential equation for heat flow in a solid

$$\nabla^2 T(r, t) + \frac{g(r, t)}{K} = \frac{\rho c}{K} \frac{\partial T(r, t)}{\partial t}$$

where

$g(r, t)$  = internally generated heat

$\rho$  = density ( $\text{kg}/\text{m}^3$ )

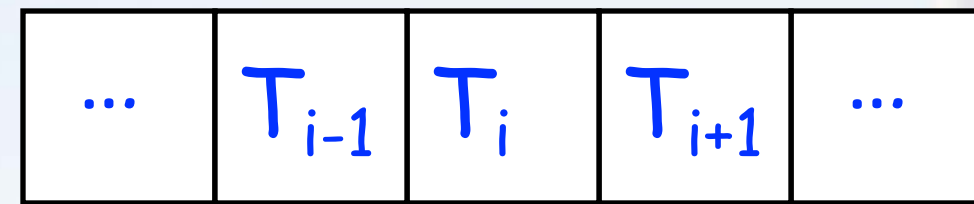
$c$  = specific heat ( $\text{J}/\text{kg}^\circ\text{K}$ )

$K/\rho c$  = thermal diffusivity



# Simple Analytical Conduction Model

- Heat flowing from (i-1) into (i)



$$Q_{in} = -KA \frac{T_i - T_{i-1}}{\Delta x}$$

- Heat flowing from (i) into (i+1)

$$Q_{out} = -KA \frac{T_{i+1} - T_i}{\Delta x}$$

- Heat remaining in cell

$$Q_{out} - Q_{in} = \frac{\rho c}{K} \frac{T_i(j+1) - T_i(j)}{\Delta t}$$

# Finite Difference Formulation

- Time-marching solution

$$T_i^{n+1} = T_i^n + d(T_{i+1}^n - 2T_i^n + T_{i-1}^n)$$

where

$$d = \frac{\alpha \Delta t}{\Delta x^2} \quad \alpha = \frac{k}{\rho C_v} = \text{thermal diffusivity}$$

- For solution stability,

$$\Delta t < \frac{\Delta x^2}{2\alpha}$$



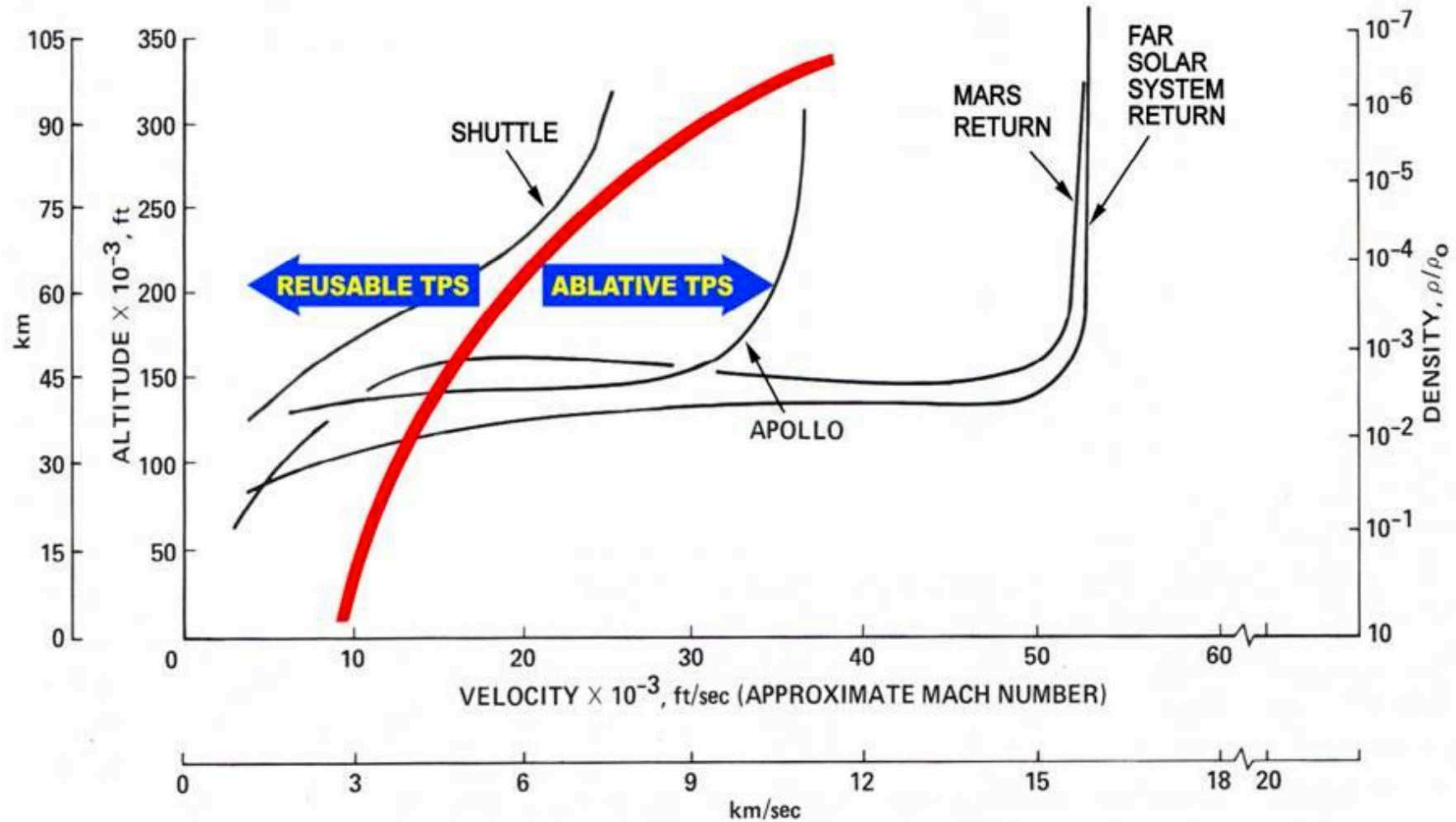
# Ablation

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- **Definition:**
  - The term ablation is encountered in many fields of science and engineering
    - In the medical field it refers to the surgical removal of a body part or tissue
    - In glaciology it refers to the removal of ice and snow from the surface of a glacier
  - In space physics, ablation is the process of absorbing energy by removal of surface material by melting, vaporization, sublimation, or chemical reaction



# Why Ablative Materials?



Courtesy Bernie Laub, NASA Ames

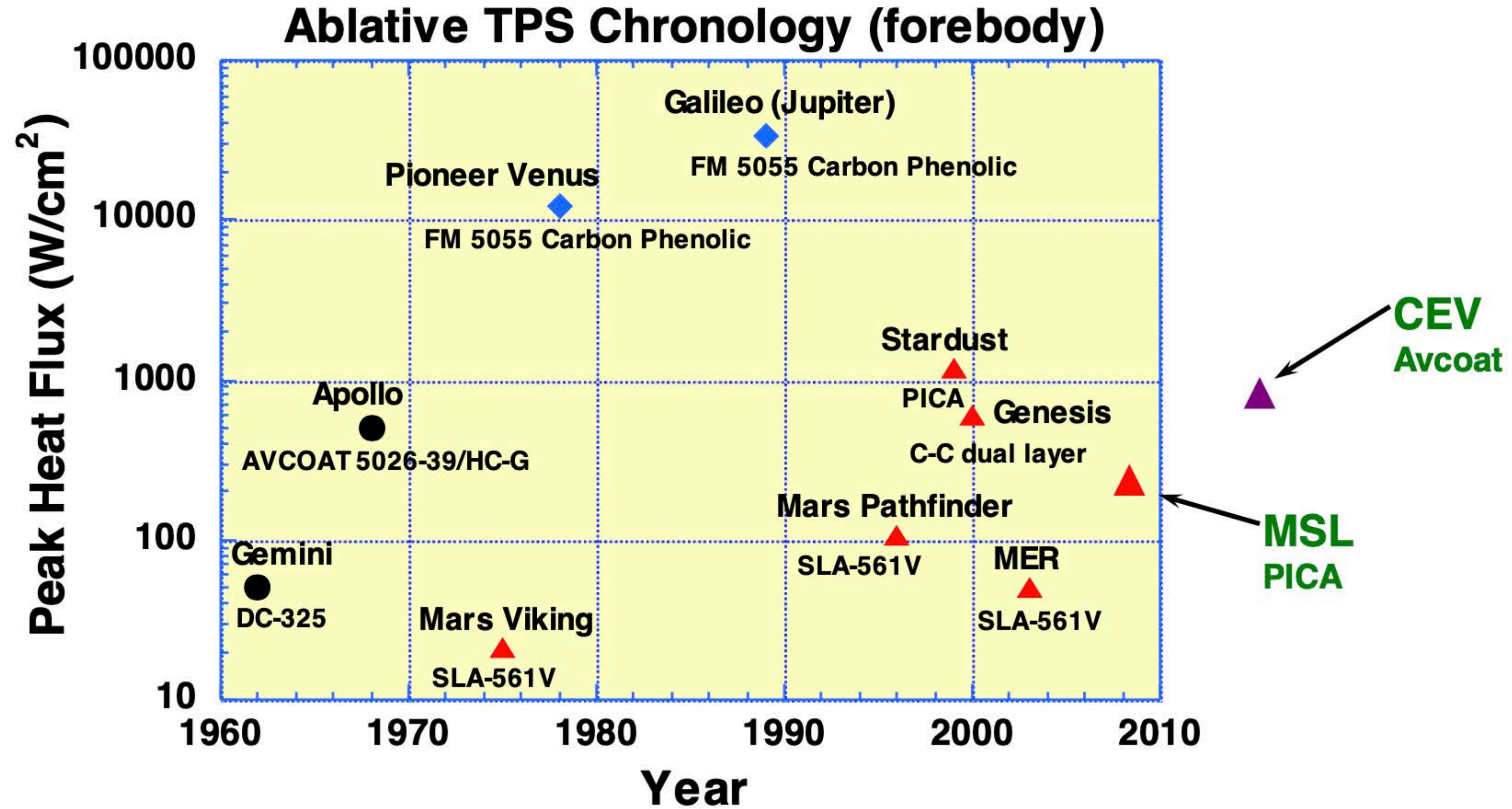
# How is TPS Chosen/Designed?

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- **Heat rate, along with pressure and shear, determine type of TPS to employ**
  - Material classes have clear performance limits marked by poor performance/material failure
- **Heat load determines overall thickness of TPS material**
- **Other design features play a role**
  - Need for tiles, forebody penetrations, compression pads, structural loads, etc. can impact material selection and TPS design
  - RF transparency for materials that protect antennae



# Ablative TPS Chronology



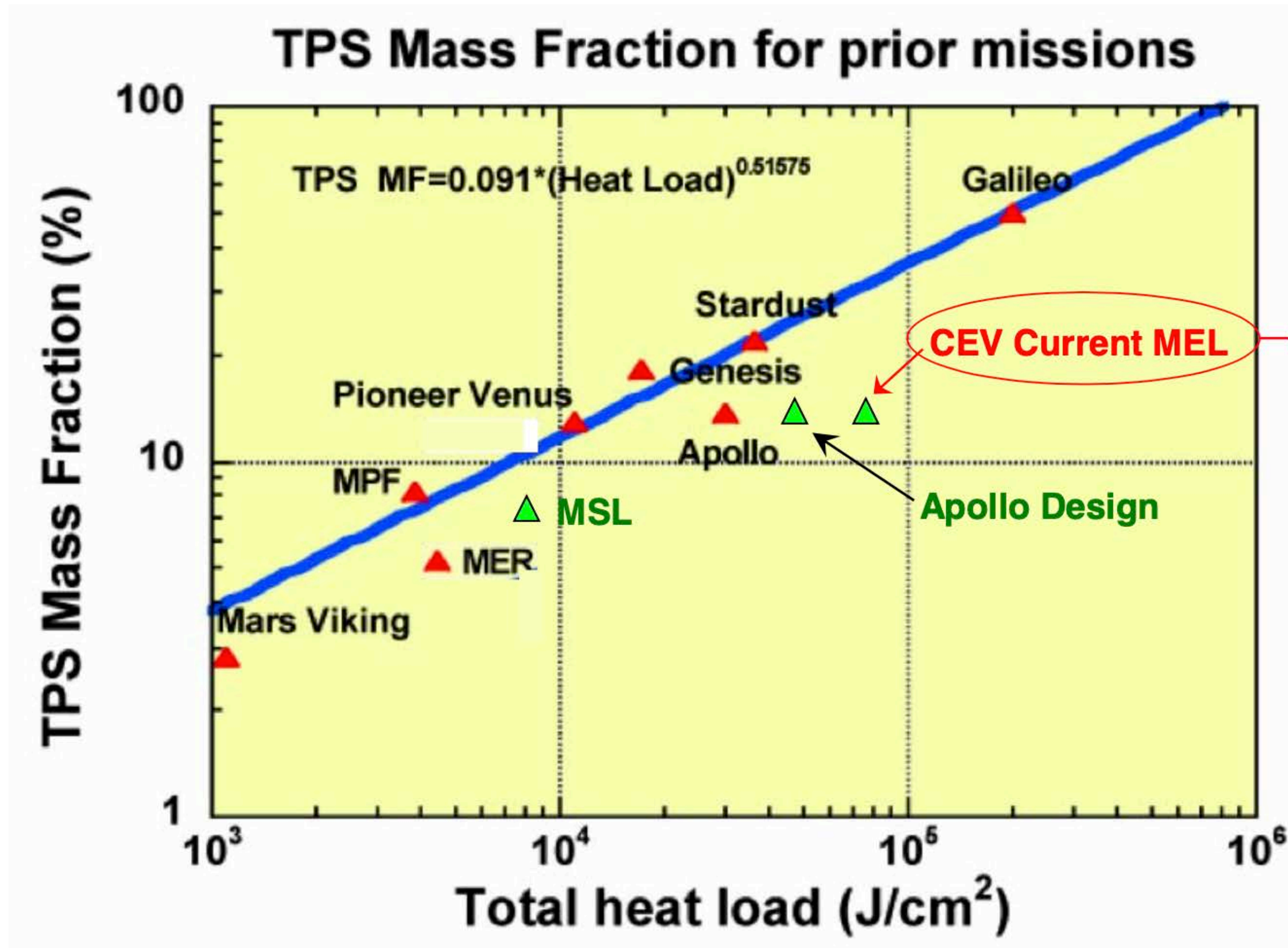


# Some Available Ablatives

Material Name	Manufacturer	Density (kg/m <sup>3</sup> )	Limit (W/cm <sup>2</sup> )	
SLA-561V	Lockheed-Martin	256	~ 200	Not viable for high shear
FM 5055 Carbon Phenolic	Fibercote (formerly US Polymeric), Hitco Inc.	1450	> 10,000	No source of heritage Rayon
MX4926N Carbon Phenolic	Cytec (pre-preg), ATK, HITCO	1450	> 10,000	Flown on Shuttle SRM, never as a heat shield
PhenCarb-20,24,32	Applied Research Associates (ARA)	320-512	~ 750	Never flown
PICA (Phenolic Impregnated Carbon Ablator)	Fiber Materials, Inc. (FMI)	265	> 1500	Must be tiled above 1m diameter
Avcoat 5026 (Apollo)	Textron Systems	513	~1000	Recreated for CEV
ACC	Lockheed-Martin	1890	~ 1500	Heavy, not readily extendible above 2m



# TPS Mass Fractions



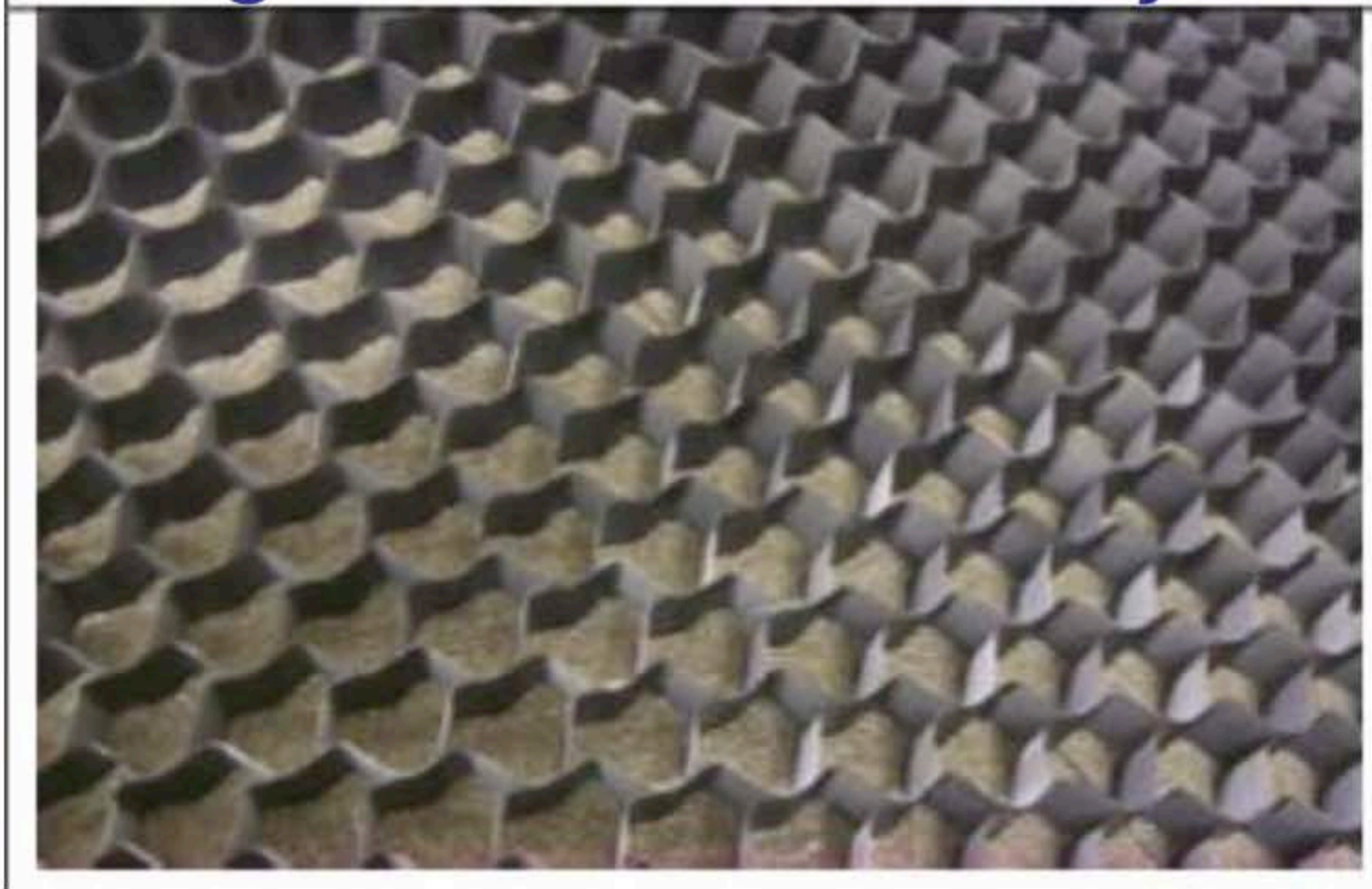
Wow! This is going to be hard w/o a significant improvement to the state of the art



# Ablative Composition

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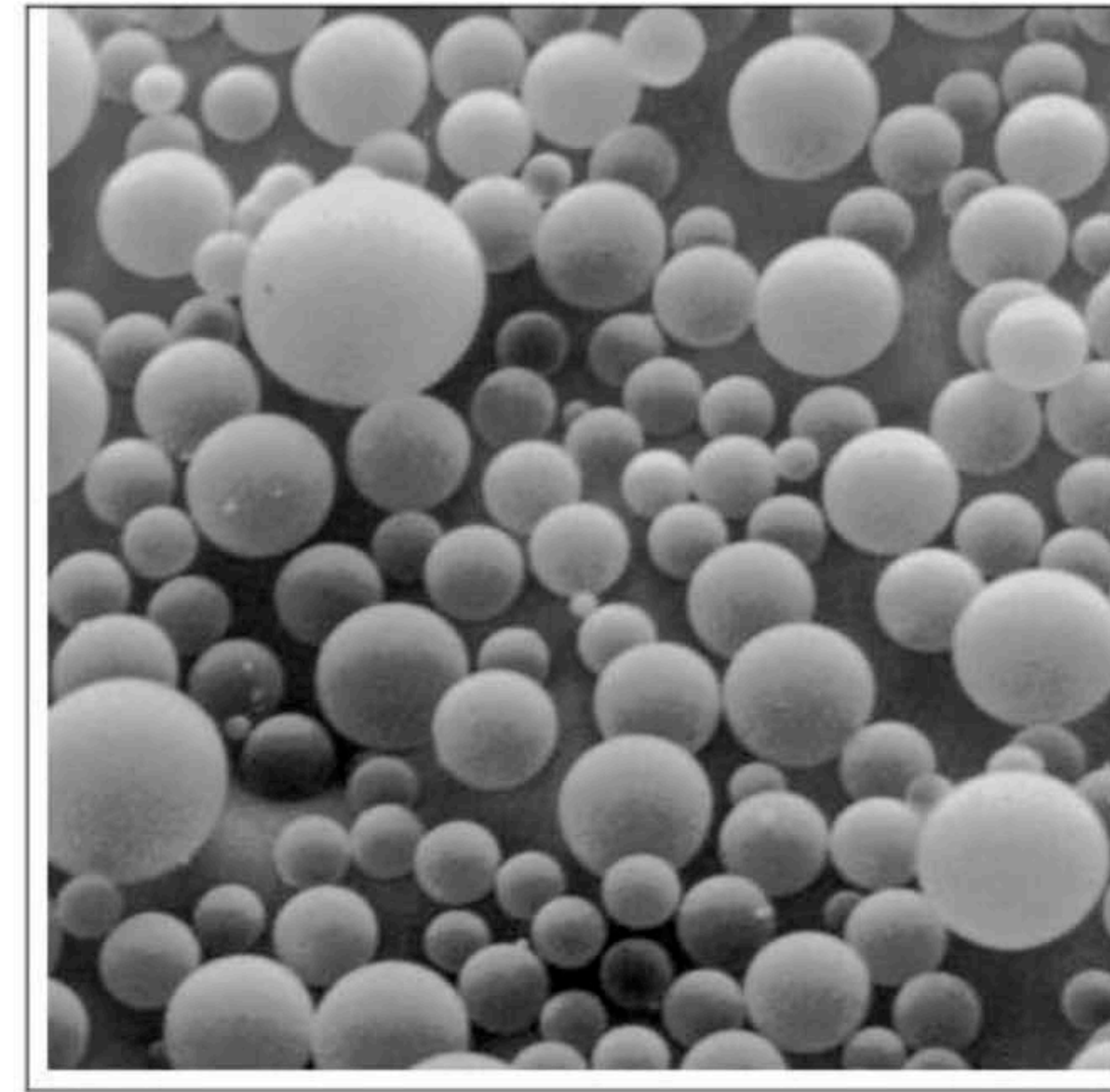
**Large Cell Phenolic Honeycomb**



**Organic Fiber Reinforced Phenolic**



**Silica Microballoons**



**Largest ~100 Microns**



# Pyrolyzing Ablators

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## **Substrate Material (e.g. fibers, cloth)**

- **Desire ability to withstand high temperatures (reradiation)**
- **Carbon is best; glass also good (heat of vaporization)**

## **Organic Resins (e.g. phenolics)**

- **Pyrolyzing ablaters only**
- **When heated resin generates gas and leaves carbon residue**
- **What are they good for?**
  - in-depth and surface transpiration
  - endothermic reactions absorb energy
  - carbon char for reradiation

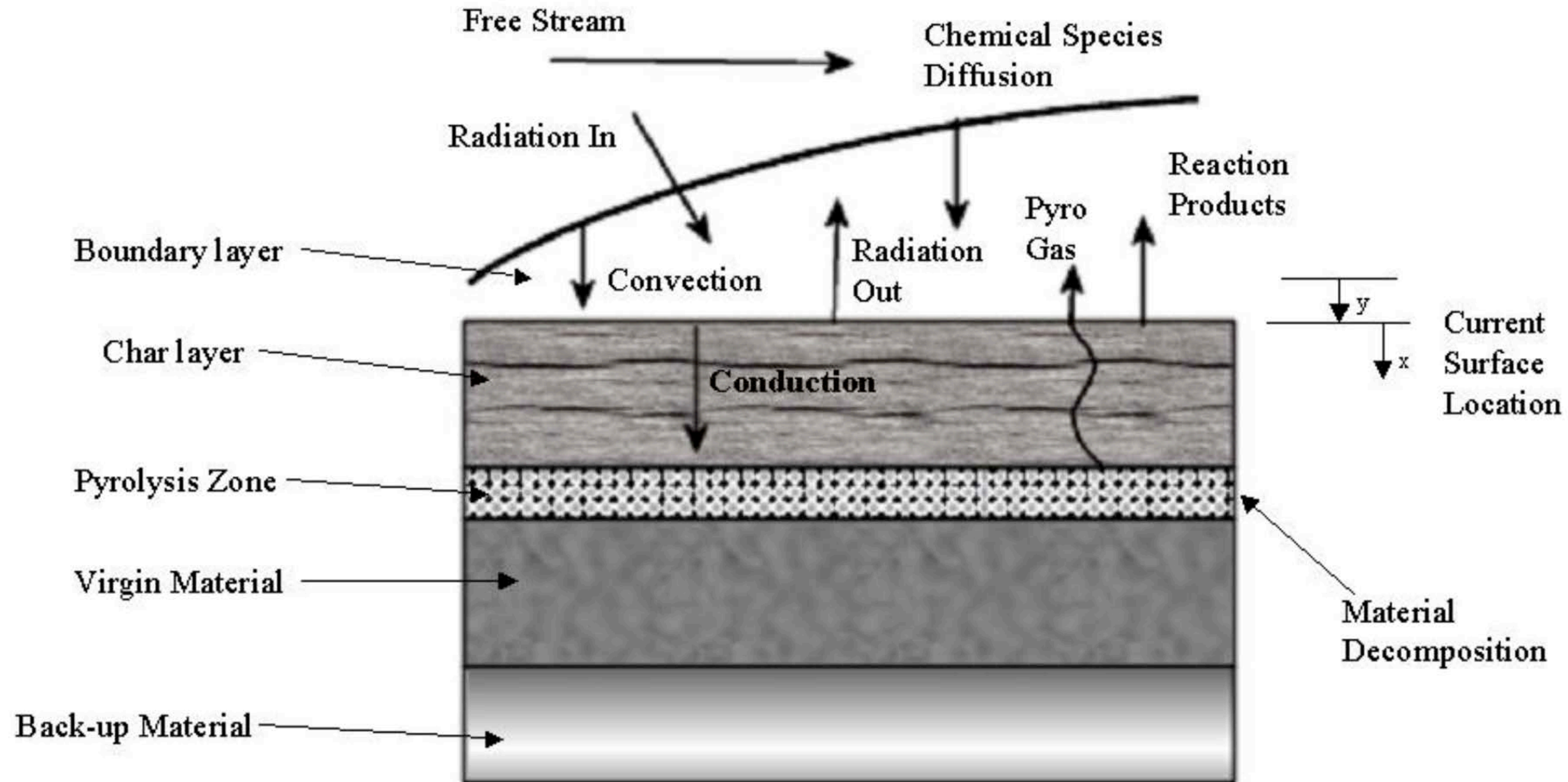
## **Additives (e.g. microballoons, cork)**

- **Density & thermal conductivity control**

## **Added Reinforcement (e.g. honeycomb)**

- **Structural integrity, bond verification (adds mass)**

# How Do Ablators Work?





# Surface Ablation Mechanisms

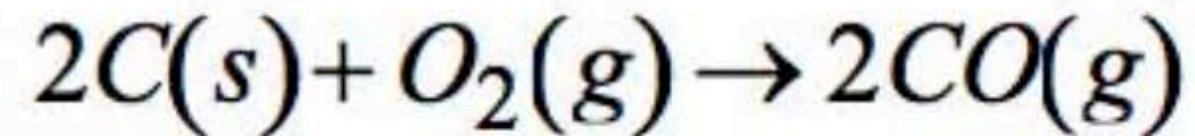
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- Melting – common ablation mechanism, but doesn't absorb much energy
- Vaporization – absorbs significant amount of energy
- Oxidation – exothermic process that adds energy
- Sublimation – Can be significant energy absorber
- Spallation – Mass loss with minimal energy absorption (Thermostructural Failure – HIGHLY UNDESIRABLE)



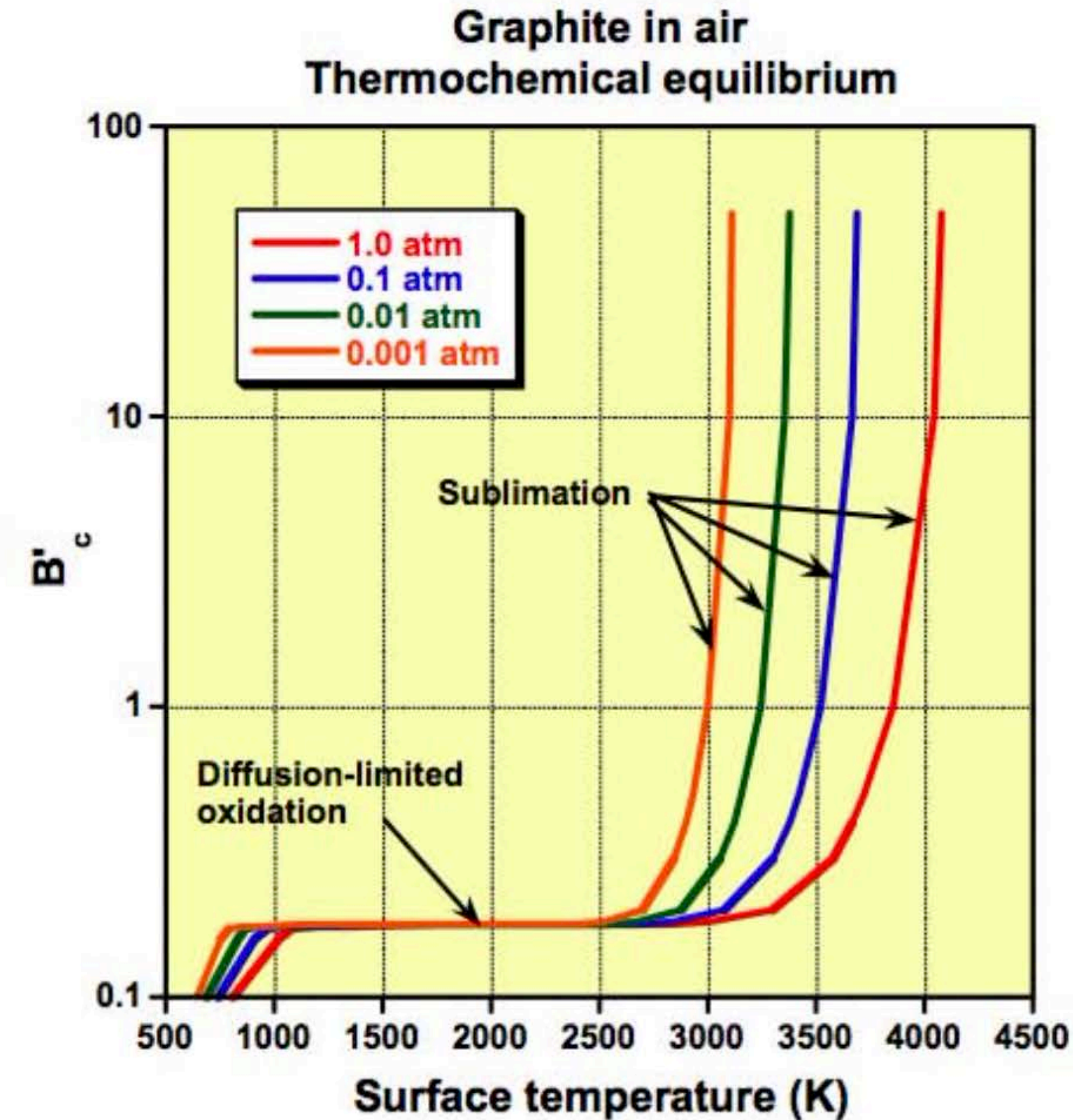
# Oxidation

- Oxidation is an *exothermic* process
- Example:



$$\Delta H_{comb} \approx -4170 \text{ kJ/g}_{carbon}$$

- Note: the B' curve for carbon in air was generated with assumptions of thermochemical equilibrium, equal diffusion coefficients, etc.
- The “equilibrium” assumption allows the diffusion-limited plateau to extend to *unrealistically low* surface temperatures



Other exothermic surface chemistry is possible (“nitridation” and “hydridation”) but these are not typically significant players



# Other Mechanisms at Play

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- Material decomposition ...aka pyrolysis
  - Endothermic reactions absorb energy
  - Convection of pyrolysis gas through the char
- Conduction through the material
  - Transfer energy to structure or heat sink
- Re-radiation from the surface
  - Largest percentage of energy is dissipated through this mechanism

# TPS Modeling Approach

- In the mid to late 1960's, Kendall, Rindal, and Bartlett, and Moyer and Rindal extended the work by Kratsch et. al.
  - Included unequal heat and mass transfer coefficients
  - Non-unity Lewis and Prandtl numbers
  - Corrected in-depth energy equation:
    - to account for the energy of the pyrolysis gas convection and generation within the solid
    - to account for grid motion due to a coordinate system that is attached to the receding surface

$$\rho c_p \frac{\partial T}{\partial t} = \frac{\partial}{\partial x_s} \left( k \frac{\partial T}{\partial x_s} \right) + (h_g - \bar{h}) \frac{\partial \rho}{\partial t} \Big|_x + \dot{S} \rho c_p \frac{\partial T}{\partial x_s} + \dot{m}_g \frac{\partial h_g}{\partial x_s} \quad (12)$$

$$-k \frac{dT}{dx} = \rho_e U_e C_H (H_{sr} - h_{sw}) + \rho_e U_e C_M \left( \sum_i (Z_{ie}^* - Z_{iw}^*) h_i^0 + B'_c h_c + B'_g h_g - B'_w h_w \right) - q^* + q_{rad, out} - \alpha q_{rad, in} \quad (13)$$

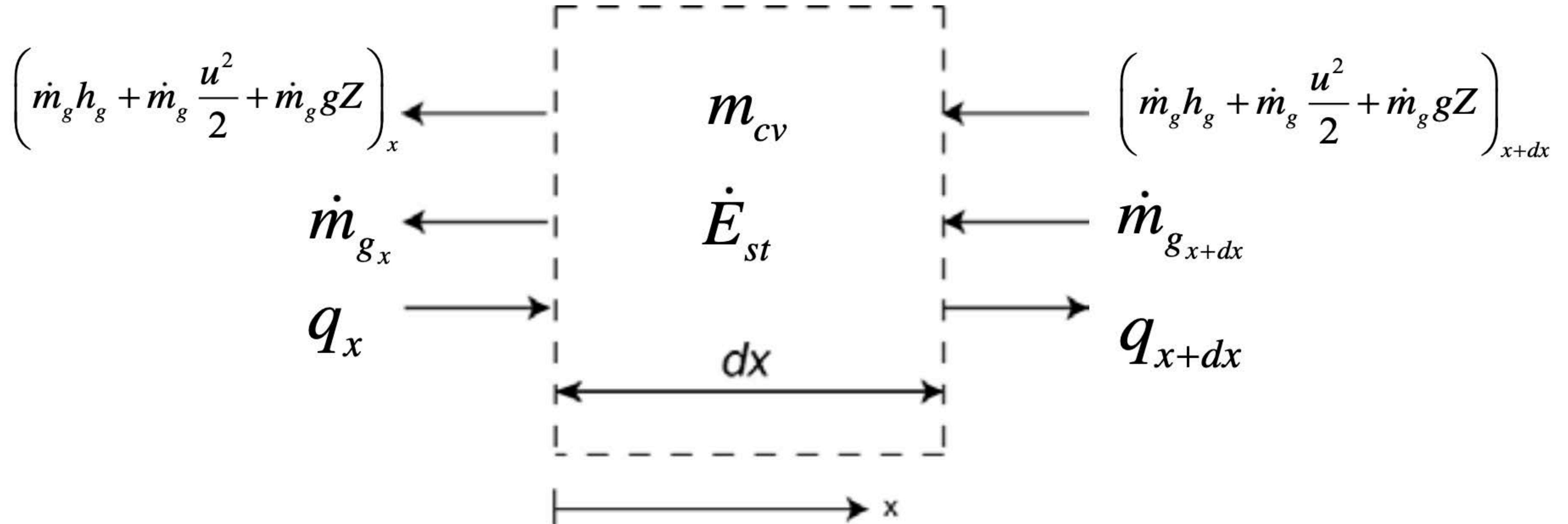
- If the diffusion coefficients are assumed equal and the Le=Pr=1.0, the surface energy balance simplifies to

$$-k \frac{dT}{dx} = \rho_e U_e C_H (H_{sr} - h_{sw} + B'_c h_c + B'_g h_g - B'_w h_w) - q^* + q_{rad, out} - \alpha q_{rad, in} \quad (14)$$



# Governing Differential Equations Derivation

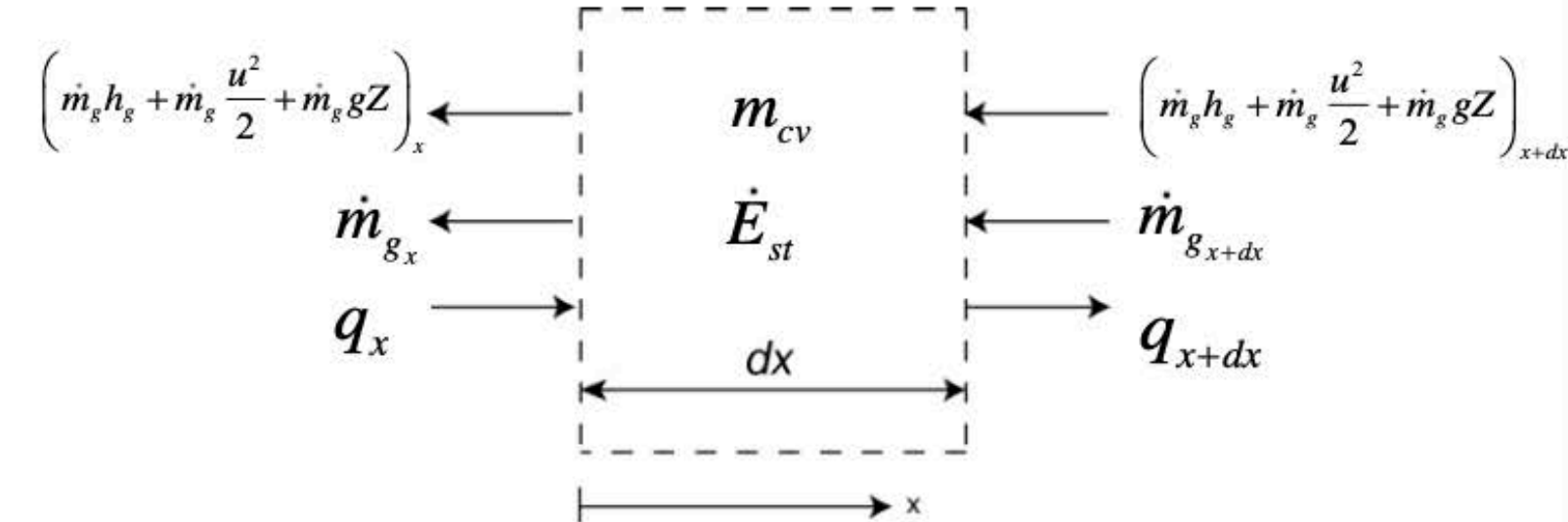
1-dimensional control volume



# Conservation of Mass

Pyrolysis gas flows from the pyrolysis zone through the porous char to the heated surface

- Assume gas flow is 1-D and normal to the heated surface
- Assume  $dp \sim 0$  across the char (neglect the momentum eqn)



$$\frac{\partial m_{cv}}{\partial t} = \dot{m}_{in} - \dot{m}_{out} \quad (15)$$

Where  $m_{cv} = \rho A dx$  (16)  $\dot{m}_{g_{x+dx}} = \dot{m}_{g_x} + \frac{\partial \dot{m}_{g_x}}{\partial x} dx$  (17)

$$A \frac{\partial \rho}{\partial t} dx = \left( \dot{m}_{g_x} + \frac{\partial \dot{m}_{g_x}}{\partial x} dx \right) - \dot{m}_{g_x} \quad \longrightarrow \quad \boxed{\frac{\partial \rho}{\partial t} = - \frac{\partial \dot{m}_{g_x}''}{\partial x}} \quad (18)$$

$\frac{\partial \rho}{\partial t}$  Determined experimentally and modeled with an Arrhenius fit

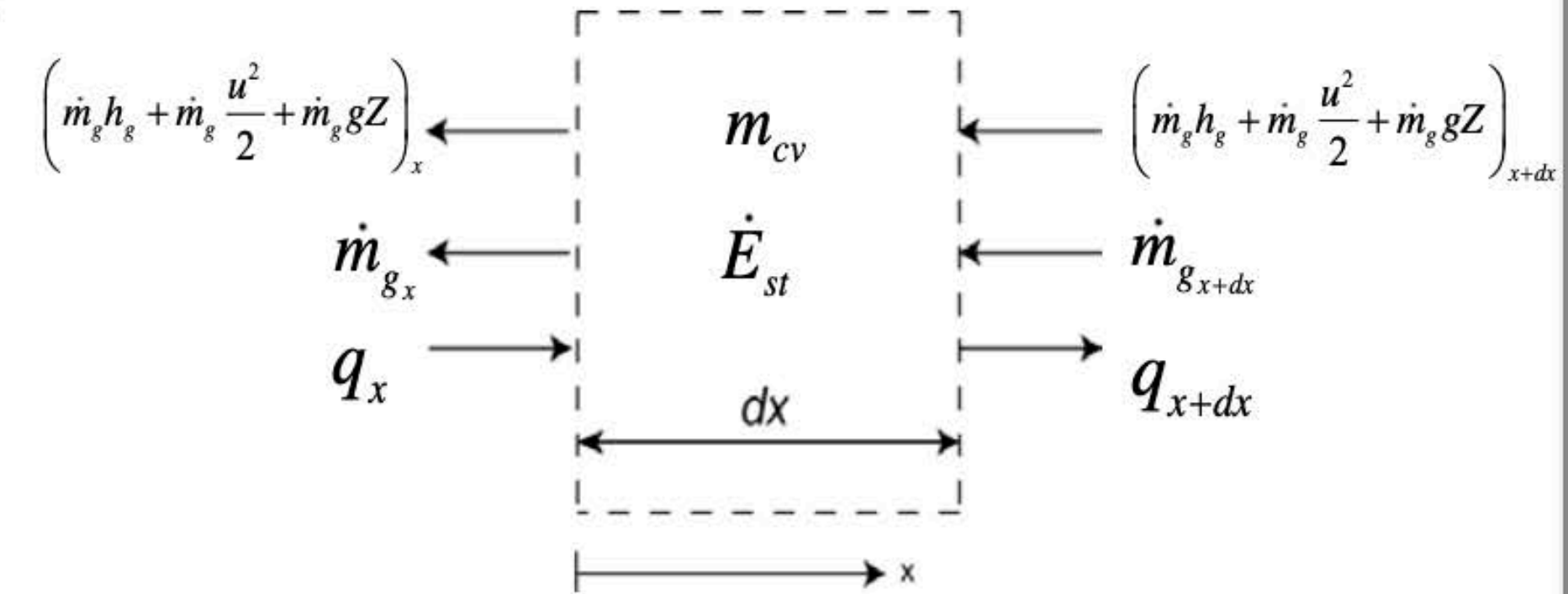
$\dot{m}_{g_x}''$  = Mass flow rate per unit area



# Conservation of Energy

- Two energies associated with this control volume

- Pyrolysis gas flow
- Heat conduction.



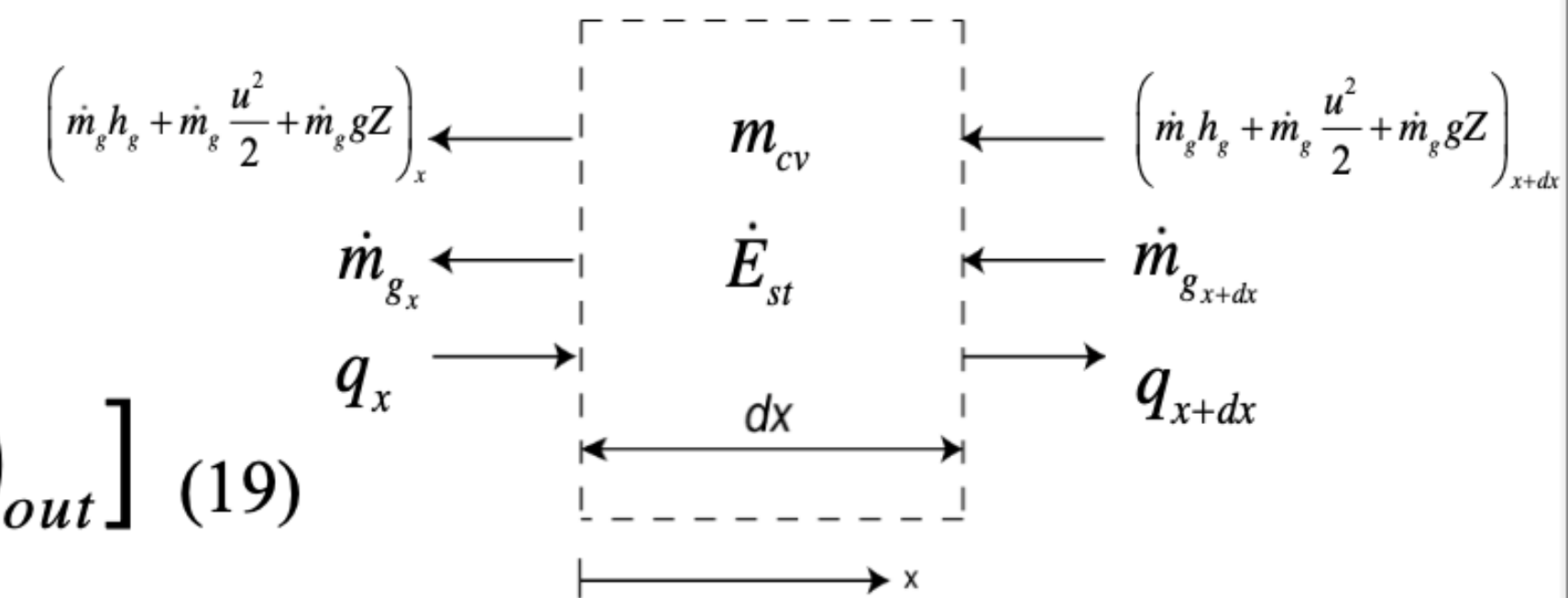
- Pyrolysis gas flow assumptions

- Pyrolysis gas is in thermal equilibrium with the charred material within the control volume
- Pyrolysis gas residence time within the control volume is small.
- Potential energy of the pyrolysis gas may be neglected since the change in height across the control volume is negligible.
- The kinetic energy of the pyrolysis gas may be neglected since it is of small magnitude relative to its enthalpy

# Conservation of Energy (2)

- 1<sup>st</sup> Law of Thermodynamics

$$\frac{dE_{cv}}{dt} = \dot{Q}_{cv} - \dot{W}_{cv} + [\dot{m}_{in}(e + Pv)_{in} - \dot{m}_{out}(e + Pv)_{out}] \quad (19)$$



Where  $e$  is the total energy per unit mass and includes kinetic, potential, and internal energy

The internal energy and flow work may be expressed in terms of the enthalpy by,  $h = u + Pv$

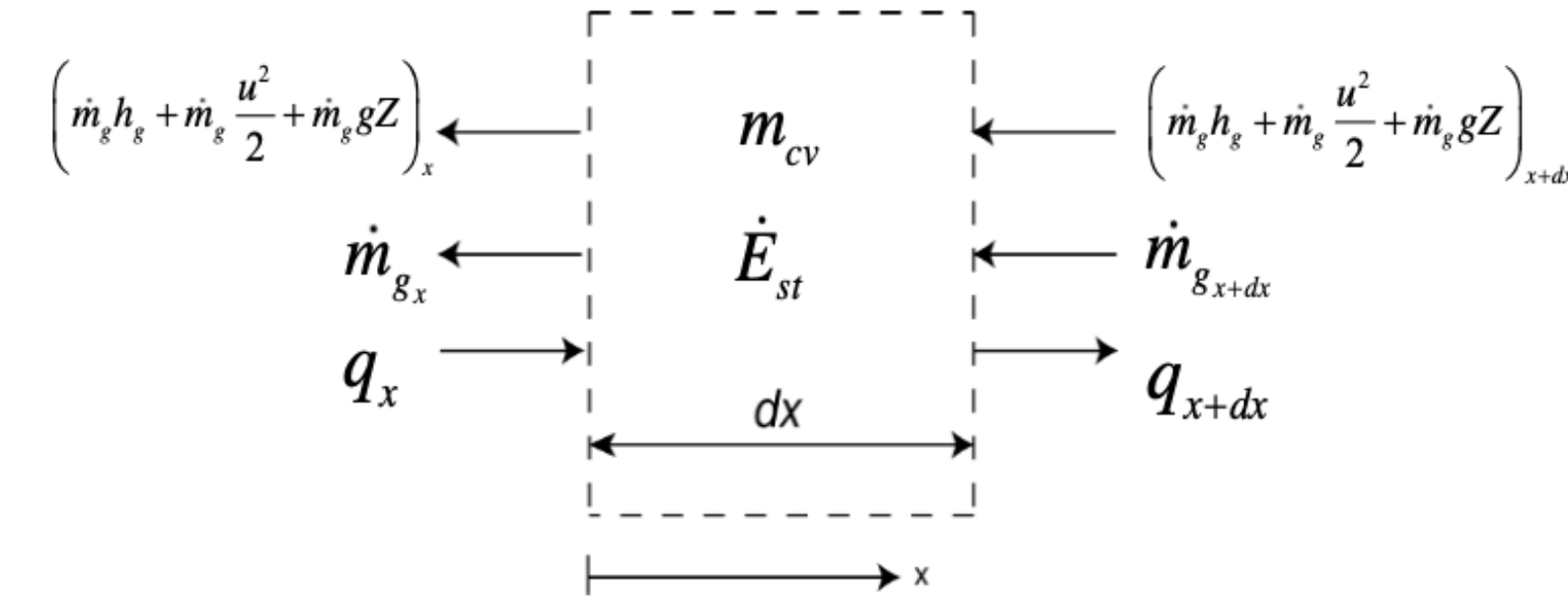
Rewriting equation 19 in a simplified form gives,

$$\frac{dE_{cv}}{dt} = \dot{E}_{in} - \dot{E}_{out} \quad (20)$$



# Conservation of Energy (3)

- The energy entering and leaving the control volume can be expressed as



$$\begin{aligned} \dot{E}_{in} &= q_x + \left( \dot{m}_g h_g \right)_{x+dx} \\ \dot{E}_{out} &= q_{x+dx} + \left( \dot{m}_g h_g \right)_x \end{aligned} \quad (21)$$

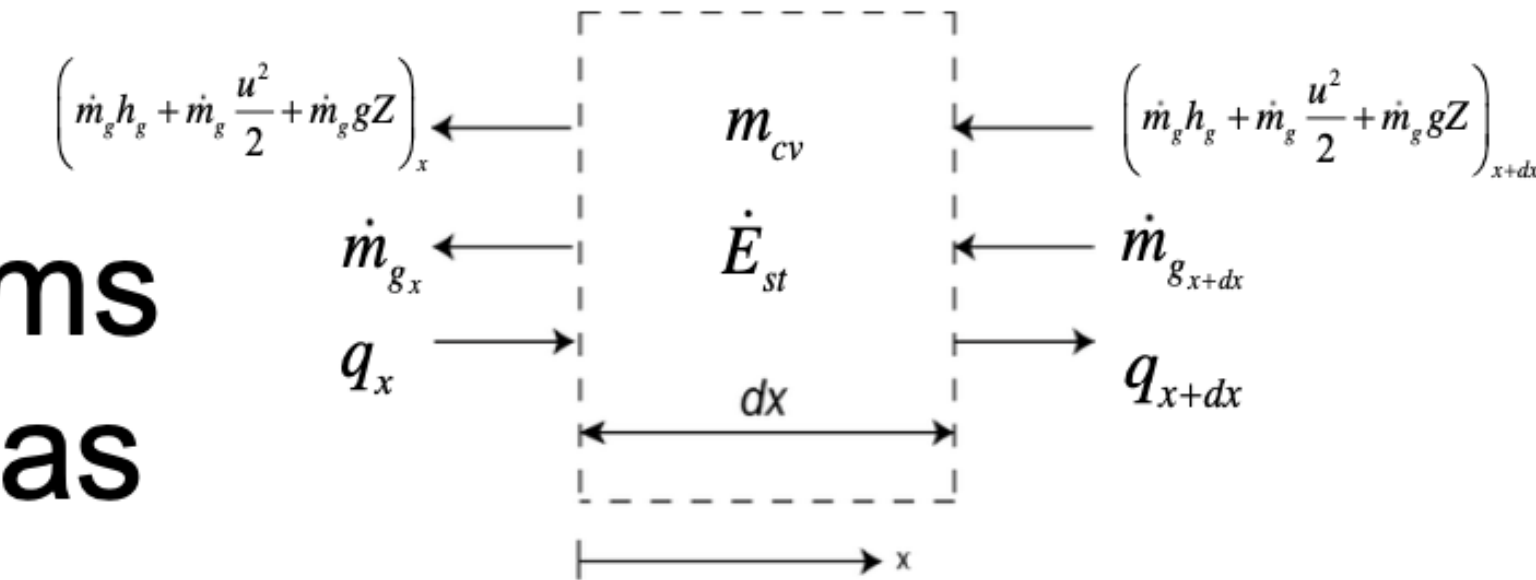
- Expressing the incremental heat conduction leaving and the convection of energy by the pyrolysis gas entering the control volume as Taylor series expansions gives, dropping

H.O.T

$$\begin{aligned} q_{x+dx} &= q_x + \frac{\partial q_x}{\partial x} dx \\ \left( \dot{m}_g h_g \right)_{x+dx} &= \left( \dot{m}_g h_g \right)_x + \frac{\partial}{\partial x} \left( \dot{m}_g h_g \right)_x dx \end{aligned} \quad (22)$$

# Conservation of Energy (4)

- The rate of energy storage within the control volume can be expressed in terms of the density and enthalpy of the solid as



$$\frac{dE_{cv}}{dt} = \frac{\partial}{\partial t} (\rho h) A dx \quad (23)$$

- Substituting eq 21 into eq 20, and using the definitions in eqns 22 and 23 gives

$$\frac{\partial}{\partial t} (\rho h) A dx = \left[ q_x + (\dot{m}_g h_g)_x + \frac{\partial}{\partial x} (\dot{m}_g h_g)_x dx \right] - \left[ q_x + \frac{\partial q_x}{\partial x} dx + (\dot{m}_g h_g)_x \right] \quad (24)$$



# Conservation of Energy (5)

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- Canceling like terms, dividing by  $A dx$ , and using Fourier's law of heat conduction eqn 24 reduces to,

$$\underbrace{\frac{\partial}{\partial t}(\rho h)}_{\text{I}} = \underbrace{\frac{\partial}{\partial x} \left( k_x \frac{\partial T}{\partial x} \right)}_{\text{II}} + \underbrace{\frac{\partial}{\partial x} (\dot{m}_{g_x} h_g)}_{\text{III}} \quad (25)$$

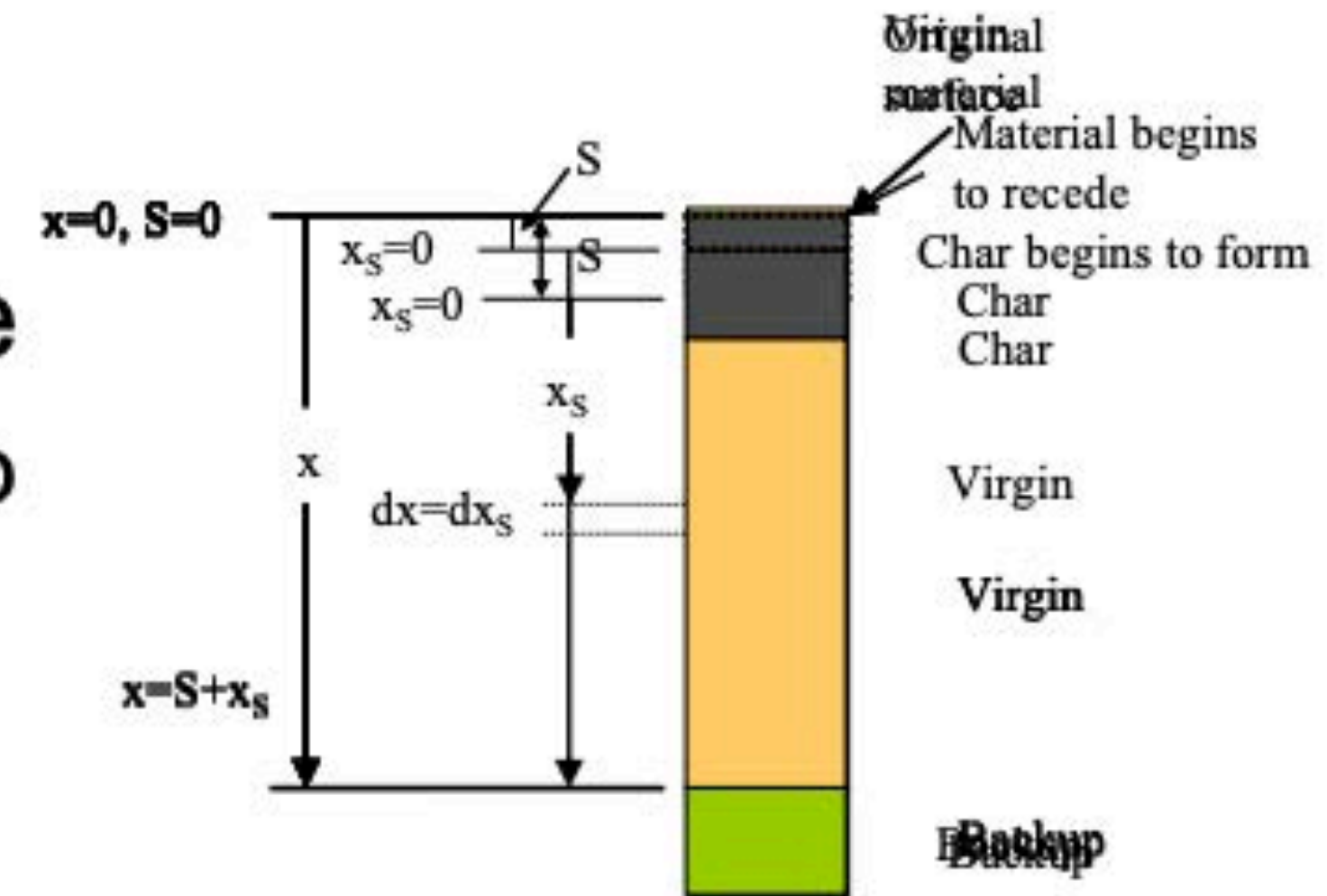
Where,

- $\rho$  : density of the solid
- $h$  : enthalpy of the solid
- $h_g$  : pyrolysis gas enthalpy
- $k_x$  : thermal conductivity in the x-direction
- $T$  : temperature
- $\dot{m}_{g_x}$  : local gas flow rate in the x-direction
- $x$  : coordinate direction

- Physically,
  - Term I represents energy storage
  - Term II represents conduction through the material
  - Term III represents convection due to pyrolysis gas flow

# Adapting to a Moving Coordinate Frame

- The control volume is not fixed in space, it is tied to the receding surface
  - Requires transforming eqns 18 and 25 into a moving coordinate system
  - After some elaborate calculus and algebraic manipulation we arrive at,



Conservation of mass in a moving coordinate system

$$\boxed{\left. \frac{\partial \rho}{\partial t} \right|_{x_s} = \dot{S} \left. \frac{\partial \rho}{\partial x_s} \right|_t + \left. \frac{\partial \rho}{\partial t} \right|_x} \quad (26)$$

Conservation of energy in a moving coordinate system

$$\underbrace{\left. \frac{\partial}{\partial t} (\rho h) \right|_{x_s}}_I = \underbrace{\left. \frac{\partial}{\partial x_s} \left( k_x \frac{\partial T}{\partial x_s} \right) \right|_t}_{II} + \underbrace{\left. \frac{\partial}{\partial x_s} (\dot{m}''_{g_x} h_g) \right|_t}_{III} + \underbrace{\dot{S} \left. \frac{\partial}{\partial x_s} (\rho h) \right|_t}_{IV} \quad (27)$$

Where terms I-III are the same as in eqn 25 and term IV is the convection of energy due to coordinate system movement



# Final Form of the Energy Equation

- It is convenient to express the  $(\rho h)$  terms in equation 27 in terms of material properties rather than the thermodynamic quantity of enthalpy
- Performing some algebra and defining a new quantity,  $\bar{h}$ , the energy equation takes the following form

$$\underbrace{\rho c_p \frac{\partial T}{\partial t}}_I = \underbrace{\frac{\partial}{\partial x_s} \left( k_x \frac{\partial T}{\partial x_s} \right)}_{II} + \underbrace{\left( h_g - \bar{h} \right) \frac{\partial \rho}{\partial t}}_{III} \Big|_x + \underbrace{\dot{S} \rho c_p \frac{\partial T}{\partial x_s}}_{IV} + \underbrace{\dot{m}''_{g,x} \frac{\partial h_g}{\partial x_s}}_V \quad (28)$$

where

$$\bar{h} = \left[ \frac{\rho_v H_v - \rho_c H_c}{\rho_v - \rho_c} \right]$$

$H_v = h_v^0 + \int_0^T c_{p_v} dT$	$\rho_v$ : virgin material density
$H_c = h_c^0 + \int_0^T c_{p_c} dT$	$\rho_c$ : charred material density
	$H_v$ : total enthalpy of the virgin material
	$H_c$ : total enthalpy of the charred material

# Final Form of the Energy Equation (2)

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$$\underbrace{\rho c_p \frac{\partial T}{\partial t}}_I = \underbrace{\frac{\partial}{\partial x_s} \left( k_x \frac{\partial T}{\partial x_s} \right)}_{II} + \underbrace{\left( h_g - \bar{h} \right) \frac{\partial \rho}{\partial t} \Big|_x}_{III} + \underbrace{\dot{S} \rho c_p \frac{\partial T}{\partial x_s}}_{IV} + \underbrace{\dot{m}_{g,x} \frac{\partial h_g}{\partial x_s}}_V \quad (28)$$

- Each term in equation 28 has physical significance
  - Term I
    - rate of sensible energy storage
  - Term II
    - net conduction through the material
  - Term III
    - creation of sensible energy due to pyrolysis (ie the heat of decomposition)
  - Term IV
    - energy convected due to coordinate system movement
  - Term V
    - energy convected away due to pyrolysis gas generation at that point



# Thermal Protection System Sizing Approach

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- **Baseline (zero-margin) sizing computed assuming nominal environments and response model to hit given bondline temperature limit**
- **Margin process then applied to account for various sources of uncertainty**
- **Appropriate factors of safety be applied to trajectory dispersions, aerothermal loads, initial conditions, and material variabilities**
- **Primary (thermal) margin is applied directly to the TPS design criterion (e.g. maximum bondline temperature)**
  - The impact of this margin on TPS thickness is material-dependent since the sensitivity of bondline temperature to thickness is material-dependent
- **Secondary (recession) margin is also employed**
  - Bondline is insensitive to excessive recession until it is too late
- **Various independent sources of error are RSS'ed to avoid stacked conservatism**
- **Additional program imposed thickness factor of safety is recommended to account for unknown unknowns**
- **Other factors (e.g. thermal stress, CTE mismatch, adhesive failure) should also be tracked as possible limiting cases**
  - Adhesive failure accounted for by maintaining conservative bondline temperature limit

# Simplified Sizing Approach

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- Using the semi-infinite solid approximation, closed-form analytical solutions to the in-depth energy equation can be derived.
- For a thick slab which has a constant surface temperature at any instant in time, the temperature at a depth  $x$  within the solid at time  $t$  is given by,

$$T(x, t) = \operatorname{erf}\left(\frac{x}{2\sqrt{\alpha t}}\right) (T_i - T_s) + T_s \quad (29)$$

where,

$T_i$  is the initial temperature

$k$  = thermal conductivity

$T_s$  is the surface temperature

$c_p$  = specific heat

$\operatorname{erf}$  is the gaussian error function

$\rho$  = density

$\alpha$  is the thermal diffusivity  $= \frac{k}{\rho c_p}$



# Simplified Sizing Approach (2)

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- In this simplified approach, the amount of material required for insulation and the amount of material required for recession are calculated separately
- To calculate the recession in an approximate way, use the data correlation parameter known as the heat of ablation ( $Q^*$ ) and solve for recession rate

$$\dot{s} = \frac{\dot{q}_{cw} \left( \frac{H_r - H_{air}^{T_w}}{H_r} \right) - \sigma \epsilon T_w^4}{\rho Q^*} \quad (5)$$

# Simple Finite Difference Approach

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- To increase the fidelity, a finite difference approximation of equation 1 can be written incorporating a simplified surface energy balance

$$\rho c_p \frac{\partial T}{\partial t} = \frac{\partial}{\partial x} \left( k \frac{\partial T}{\partial x} \right) \quad (1) \quad \dot{q}_{conv} + \alpha \dot{q}_{rad} - \dot{q}_{cond} - \epsilon \sigma T_w^4 = 0 \quad (30)$$

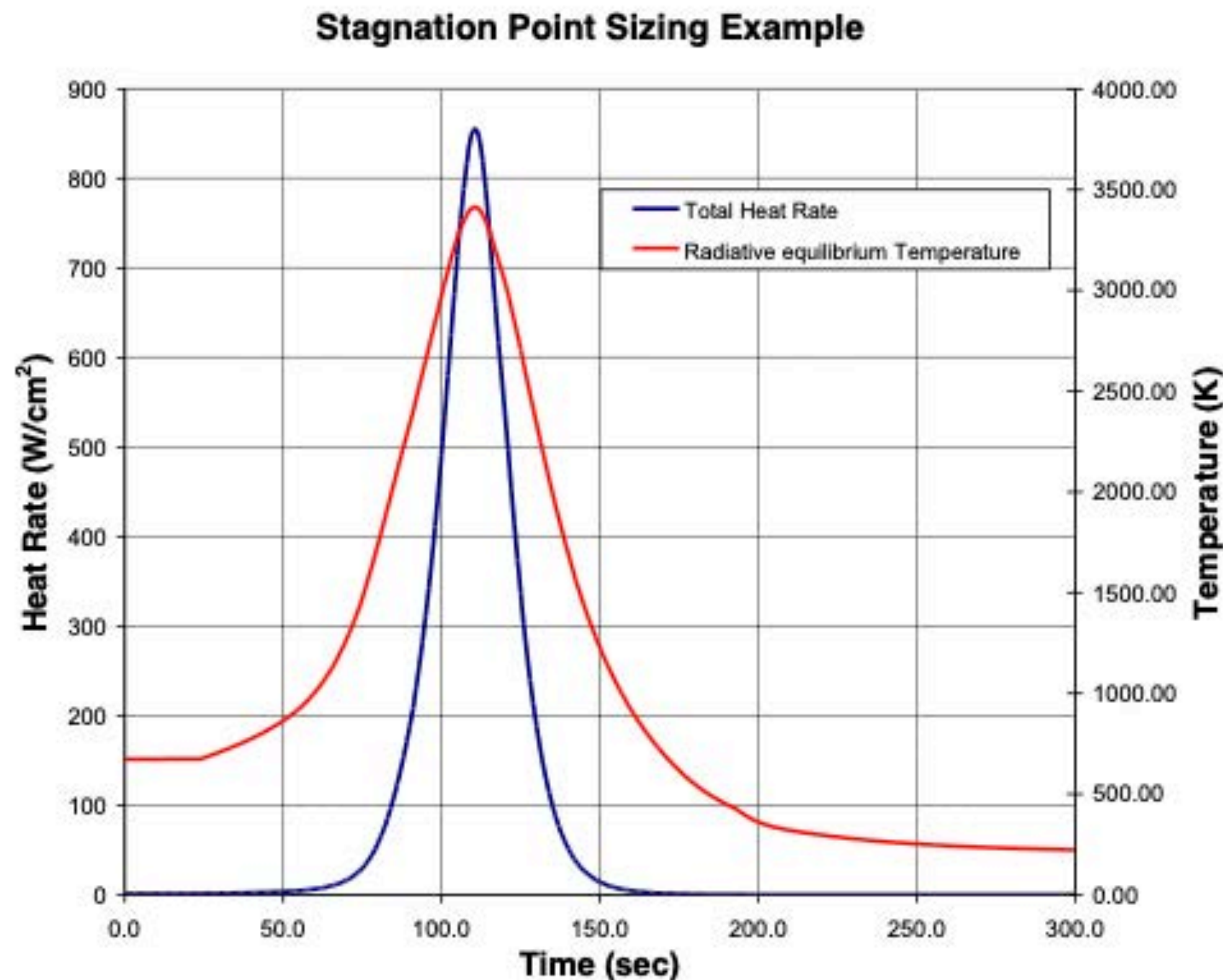
$$\left( 1 + \frac{2k\Delta t}{\rho c_p \Delta x_i^2} \right) T_i^{n+1} - \frac{2k\Delta t}{\rho c_p \Delta x_i^2} T_{i+1}^{n+1} = T_i^n + \alpha \dot{q}_{rad} \frac{2\Delta t}{\rho c_p \Delta x_i} + \dot{q}_{conv} \frac{2\Delta t}{\rho c_p \Delta x_i} - \epsilon \sigma T_i^{n4} \frac{2\Delta t}{\rho c_p \Delta x_i} \quad (31a)$$

$$T_i^n = -\frac{k\Delta t}{\rho c_p \Delta x_i^2} T_{i-1}^{n+1} + \left( 1 + \frac{2k\Delta t}{\rho c_p \Delta x_i^2} \right) T_i^{n+1} - \frac{k\Delta t}{\rho c_p \Delta x_i^2} T_{i+1}^{n+1} \quad (31b)$$



# Stagnation Point Sizing Example

- Ballistic Earth entry
  - Ballistic coefficient =  $60 \text{ kg/m}^2$  , entry velocity =  $12.6 \text{ km/s}$
  - $60^\circ$  sphere cone,  $0.8 \text{ m}$  diameter,  $r_n = 0.23 \text{ m}$
  - At the stagnation point,  $H_r$  can be approximated by  $\frac{V^2}{2}$



- PICA heat shield

$$\rho = 265.0 \frac{\text{kg}}{\text{m}^3}$$

$$k = 1.6 \frac{\text{W}}{\text{m} \cdot \text{K}}$$

$$C_p = 1592.0 \frac{\text{J}}{\text{kg} \cdot \text{K}}$$

- Radiative equilibrium temperature

$$T_w = \sqrt[4]{\left( \frac{\dot{q}_{cw}}{\epsilon \sigma} + T_{surr}^4 \right)}$$

# Stagnation Point Sizing Example (2)

- Comparing the simplified approach, the simple FD approach, and the high fidelity code CMA

